Total No. of Questions: 12]	SEAT No. :
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## **B.E.** (Mechanical)

## DYNAMICS OF MACHINERY

(2008 Pattern) (Semester - I)

Time: 3 Hours [Max. Marks: 100]

Instructions to the candidates:-

- 1) Answer any 3 questions from each section.
- 2) Answer to the two questions should be written in the separate books.
- 3) Neat diagrams must be drawn wherever necessary.
- 4) Figures to right indicate full marks.
- 5) Use of Logarithmic Tables, slide Rule, Mollier Charts, Electronic pocket Calculator & Steam tables is allowed.
- 6) Assume suitable data, if necessary.

## **SECTION - I**

- Q1) a) What do you mean by Primary and Secondary Balancing in Reciprocating Engines?[6]
  - b) Three masses A, B & C are mounted on a shaft. The planes of A and B are 100 cm apart whereas the planes of B and C are 75 cm apart. The masses A, B and C are of 30 kg, 40 kg and 32 kg and have their centre of gravity at a distance of 35 mm, 20 mm and 30 mm respectively from the shaft axis. Find the angular position of all the messes from positive x direction so that static balance is achieved.

It is required to place weights at a radial distance of 25 cm so that complete balance is achieved. If the weights are to be placed in A and C, calculate the magnitude and angular positions of desired masses. [12]

OR

- **Q2)** a) Differentiate between Static and Dynamic Balancing. Why there is need of accurate dynamic balancing of high speed machines? [6]
  - b) An air compressor has four in line cylinders at 90° intervals. The crank radius is 140 mm, while the connecting rod is 560 mm long for each cylinder. The mass of reciprocating parts is 20 kg for each cylinders and the speed of the rotation is 600 rpm. The cylinders are 300mm apart. Show that there are no out of balance primary and secondary forces and determine the corresponding magnitudes of primary and secondary couples.

Q3)	a)	Def	ine the following terms used in vibrations.	[8]	
		i)	Amplitude of vibrations		
		ii)	Resonance		
		iii)	Forced Vibrations		
		iv)	Damped Vibrations		
	b)	Wha	at is Logarithmic Decrement? Derive the relations.	[8]	
			OR		
Q4)	a)	Def	ine the following terms.	[8]	
		i)	Damping Factor		
		ii)	Coulomb Damping		
		iii)	Damping Coefficient		
		iv)	Critical Damping Coefficient		
	b)	mas is li	n under damped shock absorber is to be designed for a motor cycle ass 200 kg such that during a road bump, the damped period of vibratilimited to 2 seconds and the amplitude of vibrations should reduce e - sixteenth in one cycle. Find		
		i)	Spring Stiffness		
		ii)	Damping Coefficient of shock absorber	[8]	
Q5)	a)	Wha	at are frequency response curves? Mention the significance of the yes.	ese [6]	
	b)	in reise en percent	hachine part of mass 2 kg vibrates in a viscous medium. Determined amping coefficient when a harmonic exciting force of 35 N results a manufacture of 12.5 mm with a period of 0.2 sec. If the system excited by a harmonic force of frequency 4 Hz, what will be centage increase in the amplitude of vibration when damper is removed manufacture.  OR	ults tem the	
Q6)	a)	Exn	lain the following terms.	[6]	
2-7	-9	i)	Force Transmissibility.	r ~ 1	
		ii)	Vibration Isolation		
		,			

- b) A mass of 250 N is supported by a spring and dashpot. The spring is stretched by 150 mm due to weight and the dashpot has the coefficient of damping 1000 N per meter per sec. If the support oscillation is S.H.M. with amplitude 25 mm and frequency 6 red / sec, find. [10]
  - i) The amplitude of the Load
  - ii) The relative amplitude between Load and support.
  - iii) The amplitude of the load when the frequency of disturbing force is equal to the natural frequency.
  - iv) The amplitude of the load when the dashpot has been grounded frequency of the support is 6 rad / sec.

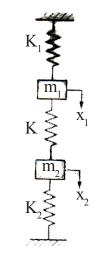
## **SECTION - II**

- **Q7)** a) Explain the Torsional vibrations of a Geared system by. [8]
  - i) Neglecting inertia of gears
  - ii) Considering inertia of gears
  - b) A rotor of 10 kg mass is mounted midway on a 2 cm diameter, horizontal shaft supported at the ends by two bearing. The bearing span is 80 cm. Because of certain manufacturing defect. The C.G. of the rotor is 0.01 mm away from its geometric center. If the system rotates at 3000 rpm, determine the amplitude of the steady state vibration and dynamic load transmitted to the bearings. [Take  $E = 2 \times 9.81 \times 10^{10} \text{N} / \text{m}^2$ ] [10]

OR

- **Q8)** a) Explain the concept of "Torsionally Equivalent Shaft". [6]
  - b) Determine the natural frequencies of the system shown in the figure.[12]

Given: 
$$K_1 = K_2 = 40 \text{ N} / \text{m}$$
  
 $K = 60 \text{ N} / \text{m}$   
 $m_1 = m_2 = 10 \text{ kg}$ 



$Q^{g}$	a)	Explain the following terms.	[4]
		i) Sound pressure level	
		ii) Sound Power level	
		iii) Acoustic Intensity	
		iv) Sound Absorption coefficient	
	b) Derive an equation that gives relation between sound Intensit Sound Pressure Level.		
	c)	A customer care containing six officers, individually makes noise level of 60, 56, 62, 53, 51 and 54 dB respectively. Add the noise levels when	
		i) All Officers are working	
		ii) When first and second officers are not working.	[6]
		OR	
Q10)	a)	Write a short note on "Sound Level Meter".	[4]
	b)	What is Sound Enclosure? Describe any one type of sound enclosure	re.[6]
	c)	Explain radiation fields of a Sound source with a Neat Sketch.	[6]
Q11)	a)	Write a short note on "Vibration Isolators".	[4]
	b)	Explain Frahm's Reed Tachometer with a neat sketch.	[6]
	c)	Explain with a neat sketch, the working principle of a centrifugal pendabsorber.	dulum [6]
		OR	
Q12)	a)	Explain piezoelectric Accelerometer with a neat sketch.	[4]
	b)	A vibrometer has a period of free vibration of 2 sec. It is attached machine with a vertical harmonic frequency of 1 Hz. If the vibromass has an amplitude of 2.5 mm relative to the vibrometer frame is the amplitude of vibration of the machine?	meter
	c)	Explain the working principle of FFT analyzer.	[6]

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